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# Emissions Characteristics of Biogas-Diesel Fuel Blends in Compression Ignition Engine

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ABSTRACT: The exceeding population around the globe is experiencing two noteworthy issues; quick consumption of regular fuel and natural debasement. Biomass derived biogas is a very promising alternative energy source because of its renewable and clean combustion characteristics compared to fossil petroleum diesel fuel. The main aim of this paper is to explore and highlight the potential of biogas-diesel dual fuel combustion mode at different engine operating conditions. Emissions characteristics of the dual fuel engine were studied at diverse energy share of biogas and compared with conventional diesel. It is inferred from the experimental results, that NO<sub>x</sub> and smoke opacity emissions were lower at all engine operating loads, while there was increase in CO, CO<sub>2</sub> and HC emissions at all energy share rates of biogas compared to diesel counterparts.

Keywords: Biogas; dual fuel; emissions; diesel; gas flow

# I. INTRODUCTION

Inflexible conservational guidelines, limited fossil fuel reserves, accelerating expenses and augmented necessitate for energy have initiated attention in supplementary progressive and innovative combustion strategies that utilizenon-conventional and alternate fuels as sources of energy. There is continuous pressure on emission control through periodically tightened regulations throughout the world. In this situation, there is an urgent need to promote use of alternative fuels as substitutes for diesel engines [1-3]. Dual fuel engines powered by biogas can be a solution to this drastic scarcity of severe power predominantly in rustic regions in India [4]. It is an attributable fact that conventional diesel engine owns the benefit of much higher thermal efficiency than SI engine at the expense of high NO<sub>x</sub> and soot emissions. Hence, the utilization of biomass derived fuel in CI engines will improve the exhalations and sustain high efficiency, which will promote the application of biomass fuel [5-7]. Moreover gaseous fuels are considered good for internal combustion engines, because of their good mixing characteristics with air. The high self-ignition temperature enables them to operate with lean mixtures and higher compression ratios, resulting in an improvement in the thermal efficiency and reduction in emissions [8-9]. Biogas is a potentially renewable, cheaply producible and environment friendly fuel. Biogas can be produced from the variety of organic substances through the anaerobic digestion process. The main constituent of biogas is methane and carbon dioxide. The composition of biogas obtained through anaerobic digestion process depends on the type of feed stocks used, and production processes evolved. Many different types of feed stocks have been explored for the production of biogas, such as cow dung, agricultural waste, animal waste, food waste, non-edible seed cakes, sewage sludge, and municipal waste [10].

Biogas is a low cetane fuel, and it cannot be directly ignited and combusted in conventional diesel engine. It requires an ignition source as the auto-ignition temperature of biogas is high [11]. Therefore, a high cetane rating pilot fuel is required to ignite and combusted the biogas, when in diesel engines. The CO2 present in the biogas acts as a diluent for the reduction of NOx tailpipe emission, when it is used in dual fuel engine [12-13]. Many research investigations have been documented on the utilization of biogas in conventional diesel engines on dual fuel mode. It was reported that, the brake thermal efficiency of the engine reduced at low to intermediate loads [14-16], while it remains unchanged, and increased at higher engine operating loads [17-19]. The NO<sub>x</sub> and smoke emissions were found to be lower [20-21]. The HC and CO concentration level were reported to be higher in some cases [22] and lower in some researches [23-24].

Karim et al. [25] investigated the performance, combustion and emissions characteristics for biogasdiesel dual fuel engines. However, their studies are mainly limited to part load or low load conditions. Nayak et al. [26] studied the emissions characteristics of a dual fuel engine operated on saw dust biomass and diesel fuel blends. They found lower NO<sub>x</sub>-smoke emissions while higher CO, HC and CO<sub>2</sub> were reported for dual fuelling operation. Significant diesel fuel saving with biomass based fuels under dual fuel mode was reported [27-30].

From the detailed literature review it can be concluded that most of the previous studies have not focused on the emission characteristics of a dual fuel engine with biogas as primary fuel and diesel as pilot fuel. For dual fuel operation, the biogas at different energy substitutions, viz., 20, 40 and 60% was inducted through the suction and diesel was injected as a pilot fuel in this experimentation. The key objectives of this investigations is to obtain biogas through anaerobic digestion of cow dung and kitchen waste and assess the

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potential of using biogas in a CI engine in view to improve emissions characteristics. Finally, the results obtained from the dual fuel operation were compared with those of diesel operation.

# **II. MATERIALS AND METHODS**

### A. Biogas production method

In the present investigation, the primary fuel used was biogas obtained by the anaerobic digestion of cow dung and kitchen waste in a fixed dome type biogas plant. A fixed dome type biogas digester comprises of a digester with a stable, non-portable gas compartment, which be situated on upper piece of the digester. Slurry was set by fraternization of water in cow's excrement in proportionate portion, and fractional amount of kitchen squander in blending supply. The slurry was then guided into the digester holder with the guide of delta compartment, where the composite carbon blends existing in the steers manure and kitchen waste breaks into easier issues by the demonstration of anaerobic organisms in the organization of water. This anaerobic breaking down of composite carbon mixes accessible in cow's fertilizer and kitchen waste secures biogas and a cycle is practiced in roughly 2 months. The biogas so shaped gathers in vault structured top of biogas generator and is provided to the motor with assistance of funnels. The devoured slurry is substituted every once in a while with new slurry to carry on the creation of biogas.

The premium diesel fuel used in the experimental work was procured from local retail petrol pump. The various physio-chemical properties of the tested fuels were in strict compliance with international ASTM standards.

### B. Experimentation

In the present investigations the single cylinder, four stroke, natural aspirated direct injection diesel engine, with a rated power output of 6.0 kW at 1500 rpm was used. All the tests were performed at standard fuel injection pressure of 210 bars and static injection timing of 26° BTDC. All experiments were performed after engine reaches the coolant temperature of 70°C at steady state condition and at a constant speed of 1500 rpm under dual fuel and single fuel mode. The detailed specifications of the test engine are summarized in Table 3. First of all, baseline data were recorded with diesel fuel for comparison. In the dual-fuel operation, biogas from the fixed dome was supplied to the flow meter and controlled amount of biogas was inducted in the inlet manifold. The biogas flow meter gives the volumetric flow of biogas into the engine cylinder. The biogas generated in the biogas plant (fixed dome type) was stored in a gas holder and supplied to the engine by a hose pipe. The gas flow rate was set at various energy levels (20, 40 and 60%) while pilot fuel supply was regulated to maintain remaining engine power. During the experiments, the engine was tested with 20%, 40%, 60%, 80% and 100% load. The exhaust gas emissions measured were CO, CO2, HC (unburned hydrocarbon), NO<sub>x</sub> and smoke opacity were measured by a gas analyser (AVL Digas 444 N) and AVL smoke opacity meter 437 C respectively.

# **III. RESULTS AND DISCUSSION**

The following section illustrated the results of emission characteristics of the diesel engine fuelled with the biogas and pure diesel were tested under different gas flow rate and engine load conditions.

### A. Hydrocarbons (HC)

The unburned hydrocarbon levels in exhaust tailpipe shows the quantity of fuel does not participate in combustion and remain unburned. The variation of hydrocarbons with engine loads for test fuels were shown in Fig. 1. Biogas fuelled dual fuel engine suffers from a higher HC emission than that of conventional diesel engine throughout the load spectrum. Same trend was observed by Tarabet et al. [31]. This may be due to the induction of biogas through the intake manifold, reduces the volume of the inducted air, and forms a fuel rich mixture zone, increases the partial burning with less oxygen [31]. The unburned hydrocarbon emissions level increases with increase in biogas fuel rate at all engine loads. At lower loads, the pilot fuel injection quantity is low, makes slower flame propagation leads to partial oxidation of fuel thereby increases the level of unburned hydrocarbon in the exhaust [32]. At 20% engine load the level of hydrocarbon under dual fuel mode (500 ppm) when compared to diesel fuel (35.5 ppm).



**Fig. 1.** Variation of HC emissions with load. At higher loads, due to higher combustion chamber temperature the level of hydrocarbon emission is lower than at lower loads under dual fuel mode. At full load engine condition the hydrocarbon level under maximum biogas flow rate were 220 ppm when compared to fossil petro-diesel (23 ppm).

#### B. Carbon monoxide (CO)

Carbon monoxide emission is the results of incomplete combustion of fuel-air mixture inside engine cylinder. The emission of carbon monoxide level in exhaust tailpipe represents the incomplete and poor fuel utilization inside combustion chamber due to low temperature of flame propagation and limited supply of oxygen. Fig. 2 shows the variation of CO with engine loads. It can be observed that the CO emission first decreases with the increase in the load up to the part load. Prakash et al. [33] also found the similar results. This is mainly attributed due to the higher cylinder gas temperature boosts the rate of combustion [33]. The CO level under dual fuel mode is considerably higher than diesel fuel at all engine loads. Due to the presence of biogas residual that dilutes the charge concentration inside engine cylinder which gives higher CO emission at low to intermediate loads. At 20% load, the concentration of CO emission is (0.32%) under dual fuel mode as compared to diesel fuel (0.30%). With increasing biogas flow rate the CO emission decreases from low load to intermediate level. At higher loads, the cylinder gas temperature is higher leads to improvement in quality of combustion thereby reduces CO level as compared to diesel mode. The CO level at full load under dual fuel mode with 3.2 kg/h was 0.21% as compared to diesel fuel (0.15%).



Fig. 2. Variation of CO emissions with load.

# C. Carbon Dioxide (CO<sub>2</sub>)

Fig. 3 shows the variation of  $CO_2$  emission with engine loads. The  $CO_2$  level in the exhaust tailpipe is higher under dual fuel mode of operation than conventional diesel fuel. It is observed that increasing biogas flow rate the  $CO_2$  level increases at all engine loads. At 20% load the concentration of  $CO_2$  level with 3.2 kg/h biogas flow rate dual fuel operation is (6.2%) when compared with diesel fuel (3.4%). This is mainly due to the dilution of  $CO_2$  present in the biogas composition which gives higher  $CO_2$  emission level [34]. It is inferred from the Fig. 5, the  $CO_2$  emission level increases with increase in engine load for all tested fuels. At full load the concentration of  $CO_2$  level with 3.2 kg/h biogas flow rate dual fuel operation is (9%) when compared with diesel fuel (6.5%).



Fig. 3. Variation of CO<sub>2</sub> emissions with load.

# D. Oxides of nitrogen (NOx)

Fig. 4 demonstrates the deviation of NOx outflows with the changing engine loads.





NOx exhalations are seen to be diminished for bi fuel mode in examination with unadulterated diesel.

Expanding mass flow rate of biogas brought about diminishing NOx emanations. The explanation behind same is abundant accessibility of CO<sub>2</sub> in biogas which aides in limiting oxygen accessibility and pinnacle ignition temperature inside the chamber which permit less measures of nitrogen and oxygen to respond [35]. Expanding engine load brings about diminished NOx exhalations. NOx emanations are noted most elevated for customary diesel and least for the greatest mass flow rate of biogas.

### E. Smoke opacity

Fig. 5 delineates the deviation of smoke opacity with the load of engine. It is seen that smoke emanation is straightforwardly in extent with engine load.





During the dual fuel activity, smoke discharges are diminished in correlation with gauge diesel and it further decline with the expansion in mass flow rate of biogas. This can be attributable to the existence of methane in biogas which helps in getting less measure of smoke [36].

# **IV. CONCLUSIONS**

An experimental study was performed by utilizing a dual fuel engine and varying the energy share of the biogas. The present experimental investigations resulted in lower emissions of NO<sub>x</sub>-smoke relationship trade-off under dual fuel mode operated on biogas-diesel fuels. However, higher CO, CO<sub>2</sub> and HC emissions were reported at all operating loads. Hence, biogas obtained from anaerobic digestion from agricultural wastes is a viable clean fuel substitute to conventional fossil petroleum diesel and hereby eliminating the harmful and hazardous emissions from diesel engine without major modifications and engine related issues. For future scope the performance and combustion characteristics of the dual fuel engine may be taken into account.

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